

Three dimensional simulation of magnetic field effect on natural convection of nanofluid

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Abstract— In the present work three-dimensional of natural convection heat transfer of water and nanofluid (Al_2O_3) in a differentially-heated cubic enclosure has been investigated numerically. The effects of fraction volume, Rayleigh number, Hartman number and inclination angle on natural convection heat transfer are analyzed. Method of solution is based on the finite volume method and an accelerated multigrid which has been tested and compared with previously published work on the basis of special cases and proved excellent agreements. The average Nusselt number increases with the increase of nanoparticles volume fraction at $Ra=10^5$ for Al_2O_3 water nanofluid. It is observed that the applying magnetic field results in a force opposite to the flow direction that leads to drag the flow and then reduces the convection currents by reducing the velocities. It has been noticed that the flow is affected by the number of Hartman and the inclination of the angle.

Keywords—natural convection, Heat transfer, Magnetoconvection, Nanofluids, three-dimensional, inclination angle.

I. INTRODUCTION

The natural convection of a conductive fluid in a closed cavity subjected to a magnetic field represents an adequate subject for scientific research. It is the challenge of many researchers during the past few decades. Natural convection is an effective procedure for many practical applications such as energy conservation, low noise and operating costs. Industries requiring natural convection are diverse. By way of example, mention may be made of nuclear reactors, cooling of electronic components, heat exchangers and solar collectors [1, 2, 3].

Natural convection in differentially heated inclined cavities was the subject of an important number of researches. To study the effect of this inclination on heat transfer and fluid flow, authors choose different ranges of angles.

Ravnik et al. [4] studied on the flow and heat transfer characteristics of the natural convection nanofluid flows in closed cavities. The simulations performed for Rayleigh number and three types of water-based nanofluids by using a three dimensional boundary element method based on flow

solver. They have shown that the use of water-based nanofluids instead of pure water improves heat transfer.

Sheikholslami and Ellahi [5] used the Boltzmann grid to study nanofluid natural convection inside a 3D enclosure in the presence of magnetic field. The Brownian motion was taken into account in the model used for the nanofluid coefficient of conductivity and viscosity. According to their results, magnetic field had a significant impact on natural convection heat transfer. In addition, the fluid convection decreased by increasing the Hartmann number. They also reported that, at a Rayleigh number of 10^5 , the heat transfer rate is highly dependent on the Hartmann number. However, this dependency is decreased at smaller Rayleigh numbers.

Zhou et al. [6] performed a three-dimensional lattice Boltzmann simulation for mixed convection for nanofluid filled enclosure in presence of magnetic force. The influences of Rayleigh number, solid volume fraction of nanofluid, Hartmann number and Richardson number on the fluid flow and heat transfer are studied. They showed that the Adding nanoparticles of Al_2O_3 into pure water improve natural convection heat transfer in a cubic cavity. However, the effect of convective heat transfer enhancement is more pronounced at low Rayleigh numbers. The enhancement will be weakened and even reversed at high Rayleigh numbers. (2) In contrast to Rayleigh number, the increase of Hartmann number decreases the heat transfer rate. This effect is more pronounced at high Rayleigh numbers. In addition, the influences of external magnetic field on heat transfer vary with different orientations.

Kolsi et al. [7] performed a computational study for 3D MHD natural convection inside a cubical enclosure with an inclined plate. They found an optimal inclination angle for the plate. The maximum heat transfer is formed when $h = 180^\circ$ but minimal value of average Nusselt number is changed according to nanoparticle addition into base fluid. Also, a minimum heat transfer value is formed at $h = 270^\circ$ almost for all cases. But effects of inclined plate became clearer for higher values of Rayleigh number. Heat transfer increases with increasing of Rayleigh numbers

Krunul and Gangawane [8] made a study on natural convection in a partially heated open ended square cavity subjugated to a magnetic field by using thermal lattice Boltzmann method (TLBM) based on single relaxation time (SRT) method. He showed that cavity with the applied

magnetic field at $Ha=45$ offers highest heat transfer restriction than other considered cases.

Mahian et al. [9] performed a theoretical and experimental study on the Natural convection of silica nanofluids in square and triangular enclosures. Results indicate that the average Nusselt number could be estimated theoretically with the same trend and maximum difference of 4.5%.

Kolsi et al. [10] investigated the combined buoyancy-thermocapillary convection in 3D enclosure filled with Al_2O_3 nanofluid and showed that the increase of nanoparticle volume fraction causes heat transfer enhancement.

Purusothaman et al. [11] made a study on the 3D natural convection in a cubical enclosure with presence of thermally active heater and external magnetic field. The results show that the heat transfer rate and flow depended strongly on the strength of the magnetic field.

Bondareva et al. [12] investigated a numerical study on the natural convective heat transfer combined with melting in a cubical cavity filled with a pure gallium under the effects of inclined uniform magnetic field and local heater. It is mounted that an increase in the Hartmann number leads to suppression of the convective flow and heat transfer.

Al-Rashed et al. [13] studied the effect of magnetic field on natural convection inside a cubical cavity filled with CNT-water nanofluid. They found that for all the Rayleigh numbers the Bejan number increase by increasing nanoparticles volume concentration. Also, they mounted that the effect of angle of inclination on the total entropy generation is more sensible when $Ha = 50$. Alsaady and al. [14] they carried out a bibliographic study on thermo-physical properties and thermo-magnetic convection of ferrofluid. They found that the thermal conductivity of a ferrofluid can reach 300 times that of the base fluid. Gui and al. [15] performed an experimental study for investigated the effect of various water based ferrofluid types on the heat transfer properties under the application of an external magnetic field. The results obtained showed that a higher solid volume concentration of magnetic nanoparticles enhances heat transfer rates. However, they observed that the effective thermal conductivity of the ferrofluid is reduced because of magnetization of the nanoparticles. Also, they observed that heat transfer rate was diminished with increasing magnetic field strength.

II. PHYSICAL MODEL AND NUMERICAL APPROACH

A. Physical Model

As shown in Figure 1, a three-dimensional cubic enclosure of side length L filled with Al_2O_3 -water nanofluid is considered ($Pr=6.2$). The right sidewall of the enclosure is maintained at a constant hot temperature T_H , while the opposite wall has a constant cold temperature T_C . Four other walls of the enclosure are adiabatic. The nanofluid is assumed to be Newtonian, incompressible and the flow is laminar. The thermophysical properties of the base fluid (water) and Al_2O_3 nanoparticles are given in Table 1. The thermophysical properties of the nanofluid are taken to be constant except for the density

variation in the buoyancy force, which is estimated by using the Boussinesq approximation. An external magnetic field is located at the center of the left hot wall which induced the magnetic convection at an angle of γ .

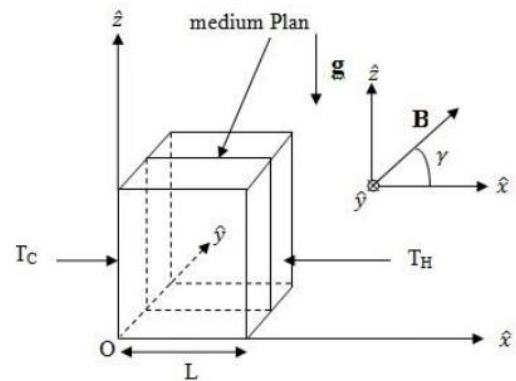


Fig. 1. Physical model

Table.1: Thermophysical properties of water and Al_2O_3 nanoparticles [16].

	Pure water	Al_2O_3
$\rho(Kg\ m^{-3})$	997.1	3,970
$C_p(JKg^{-1}K^{-1})$	4179	765
$\nu(m^2\ .s^{-1})$	0.613	40
$\beta(K^{-1})$	21×10^{-5}	85×10^3
$K(W\ m^{-1}\ K^{-1})$	1.74×10^7	131.7×10^7
$\sigma(\Omega^{-1}\ m^{-1})$	0.05	10^{-12}

B. Numerical approach

Based on the above assumptions, the governing dimensionless equations for conservation of mass, momentum, and energy of the three-dimensional unsteady nanofluids magnetoconvection (MHD) flow can be written as follows:

Conservation of mass equation:

$$\frac{\partial U}{\partial X} + \frac{\partial V}{\partial Y} + \frac{\partial W}{\partial Z} = 0 \quad (1)$$

Momentum equation:

Projection according to (ox):

$$\frac{\partial U}{\partial \tau} + U \frac{\partial U}{\partial X} + V \frac{\partial U}{\partial Y} + W \frac{\partial U}{\partial Z} = -\frac{\partial P}{\partial X} + \frac{\nu_{nf}}{\nu_f} Pr \left(\frac{\partial^2 U}{\partial X^2} + \frac{\partial^2 U}{\partial Y^2} + \frac{\partial^2 U}{\partial Z^2} \right) + \frac{\sigma_{nf}}{\sigma_f} Pr Ha^2 \sin(\gamma) (-U \sin(\gamma) + W \cos(\gamma)) \quad (2)$$

Projection according to (oy):

$$\frac{\partial V}{\partial \tau} + U \frac{\partial V}{\partial X} + V \frac{\partial V}{\partial Y} + W \frac{\partial V}{\partial Z} = -\frac{\partial P}{\partial Y} + \frac{\nu_{nf}}{\nu_f} \text{Pr} \left(\frac{\partial^2 V}{\partial X^2} + \frac{\partial^2 V}{\partial Y^2} + \frac{\partial^2 V}{\partial Z^2} \right) + \frac{\sigma_{nf}}{\sigma_f} \text{Pr Ha}^2 V \quad (3)$$

Projection according to (oz):

$$\frac{\partial W}{\partial \tau} + U \frac{\partial W}{\partial X} + V \frac{\partial W}{\partial Y} + W \frac{\partial W}{\partial Z} = -\frac{\partial P}{\partial Z} + \frac{\nu_{nf}}{\nu_f} \text{Pr} \left(\frac{\partial^2 W}{\partial X^2} + \frac{\partial^2 W}{\partial Y^2} + \frac{\partial^2 W}{\partial Z^2} \right) - \frac{\beta_{nf}}{\beta_f} Ra \text{Pr} \theta + \frac{\sigma_{nf}}{\sigma_f} \text{Pr Ha}^2 \cos(\gamma) (U \sin(\gamma) - W \cos(\gamma)) \quad (4)$$

Energy equation:

$$\frac{\partial \theta}{\partial \tau} + U \frac{\partial \theta}{\partial X} + V \frac{\partial \theta}{\partial Y} + W \frac{\partial \theta}{\partial Z} = \frac{\alpha_{nf}}{\alpha_f} \left(\frac{\partial^2 \theta}{\partial X^2} + \frac{\partial^2 \theta}{\partial Y^2} + \frac{\partial^2 \theta}{\partial Z^2} \right) \quad (5)$$

In order to cast the governing equations into a dimensionless form, the following dimensionless parameters are introduced:

$$Ra = \frac{g(T_H - T_C)H^3 \beta_f}{\nu \alpha}; \text{Pr} = \frac{\nu_f}{\alpha_f}; Ha = LB \sqrt{\frac{\sigma_f}{\mu_f}}$$

The average Nusselt number (Nu_{avg}) is defined in the heated wall as:

$$Nu_{avg} = -\frac{K_{nf}}{K_f} \int_0^1 \frac{\partial \theta}{\partial Y} \Big|_{Y=0} dX$$

III. RESULT AND DISCUSSION

The presented results in this work are obtained by using a finite volume home FORTRAN code, named NASIM and developed by the second author which use multi-grid solver explained in details in previous works [17].

In this section, we will adopt the multi-grid solver to study the natural convection of nanofluids in partially heated cubic enclosures with temperature-dependent properties. Numerical simulations are performed for wide ranges of different parameters:

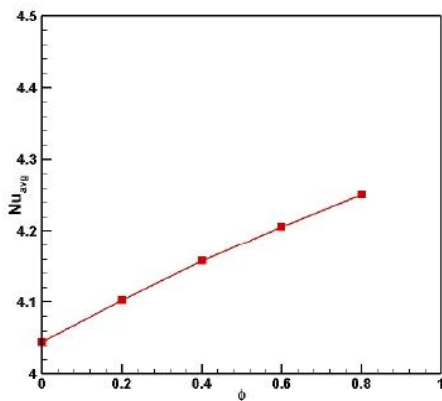


Fig.2. Variation of Nusselt number at different volume fractions.

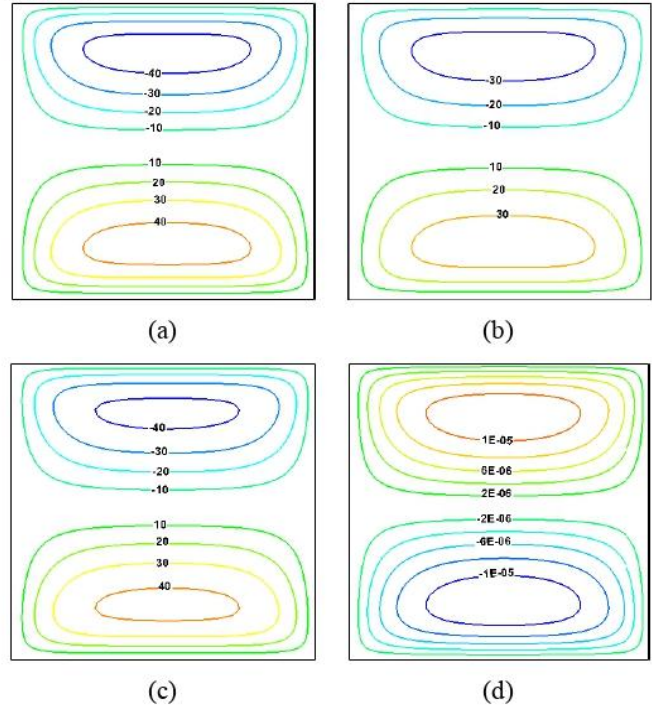


Fig.3. Comparison of the stream line velocity for various inclined angles (a) 0°, (b) 30°, (c) 60°, (d) 90° at $Ra=10^5$.

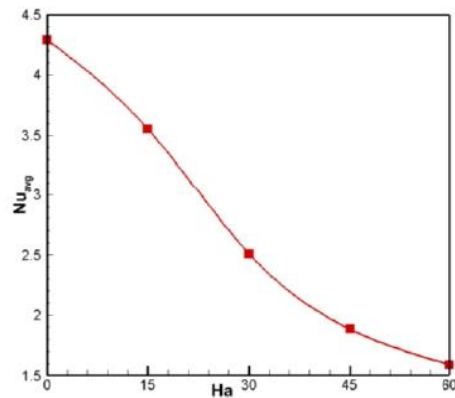


Fig.4. Comparison of the average Nusselt for various Hartman number at $Ra=10^5$ and $\gamma=45^\circ$.

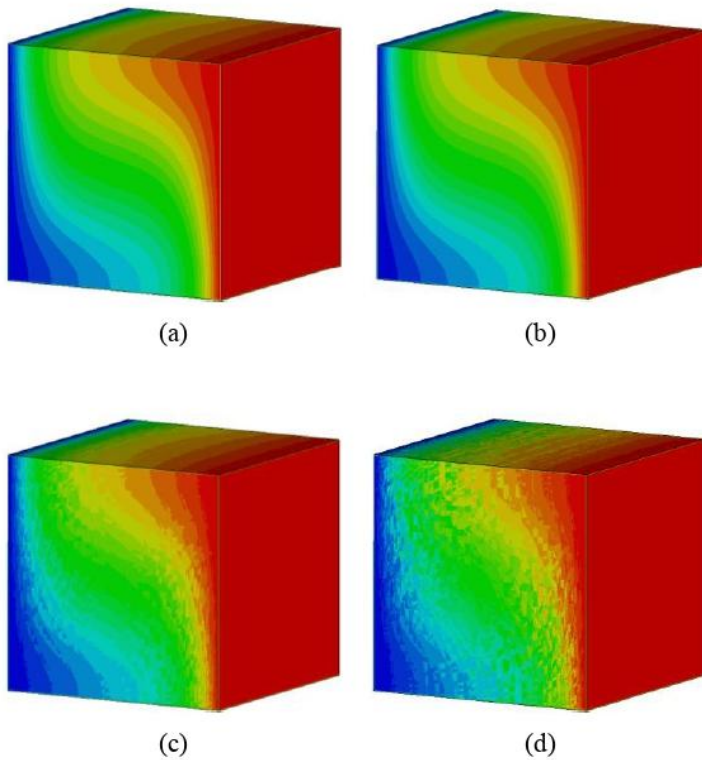


Fig.5. Effects of inclination angle (a) $Ha=0$, (b) $Ha=15$, (c) $ha=30$, (d) $Ha=60$ on isotherms ($Ra = 10^5$, $\phi=1\%$, $(\gamma)=45^\circ$).

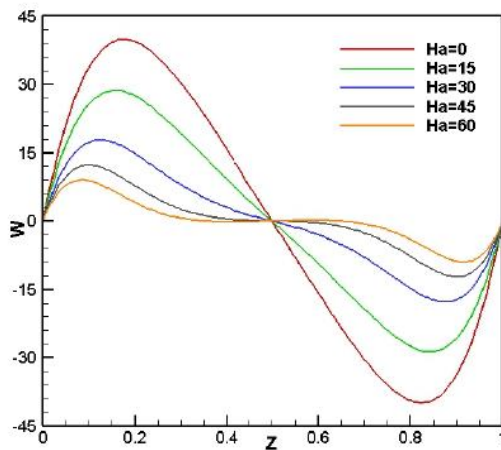


Fig.6. Velocity profiles through a centre of the $Y=0.5$ plane for natural convection in a differentially heated cubic cavity for different Hartman number.

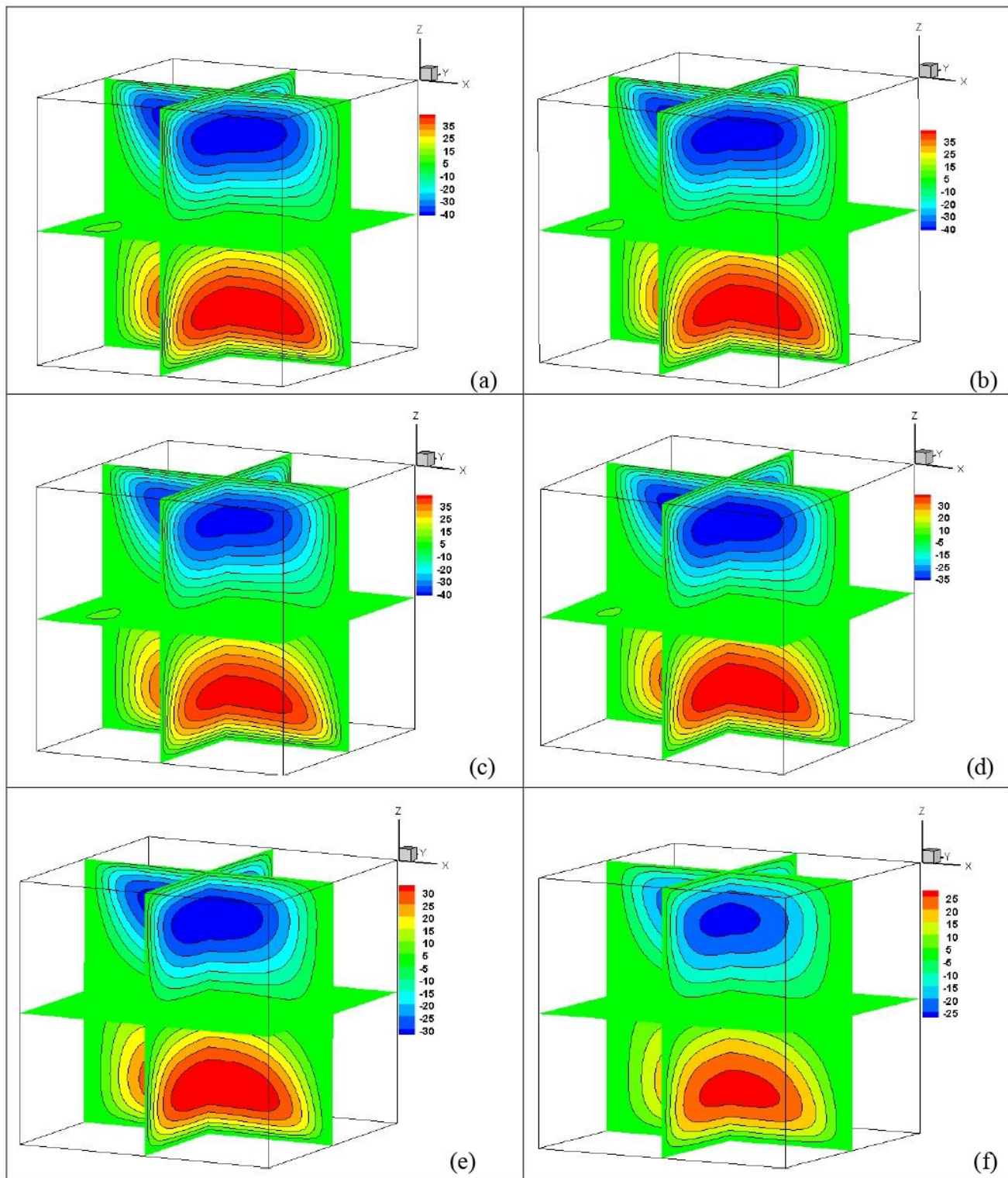


Fig.7. velocity vectors for various inclined angles (a) 0°, (b) 15°, (c) 30°, (d) 45°, (e) 60° and (f) 75° at $Ra=10^5$ and $\phi=1\%$.

In order to study the effects of the external applied magnetic field on heat transfer, a series of simulation tests were performed within a partially heated cubic cavity.

Fig.2 shows the effects of volume fractions of nanoparticles on the natural convection heat transfer of nanofluids in a cubic cavity. Four simulation cases, namely, $\phi = 0, 2\%, 4\%, 8\%$, at $Ra=10^5$ have been realized. It is observed that the addition of nanoparticles in the base fluid improves the heat exchange rates in the fluid and consequently leads to the improvement of the energy transfer.

For pure water and Al_2O_3 -water nanofluids, **fig.3** present the the stream line velocity for various inclined angles at $Y=0$ and $Ra=10^5$. it has been shown that the speed of the flow is affected by the angle of inclination.

Fig.4 illustrates the variation of average Nusselt number with Hartmann number at Rayleigh numbers 10^5 . As can be seen in the figure, the increase of

the Hartmann number decreases the heat transfer rate. It is interesting to see that this fact is not valid for $Ha=60$ and $Ha=30$. For lower Ha numbers, the magnetic forces are weaker so the flow is not totally under the influences of these forces.

Fig.5 presents the effects of Hartmann number on isotherms. It is observed the isotherms become parallel to the side wall as Hartmann number increases. It is due to the increase of Hartmann number leads to increasing Lorentz force, which results in the domination of conduction heat transfer.

The comparison of profiles for $Ra=10^5$ and $\gamma=45^\circ$ for water and nanofluids is shown in **Fig.6** the influence of the magnetic field on the velocity of the flow is observed by the change in the number of Hartman. The decreased velocity results in decreased convective heat transfer

Fig.7 presents velocity vectors for nanoparticle concentration of 1% at $Ra = 10^5$. When the inclined angles increase, the buoyancy force divide into vertical and horizontal forces throughout the cavity. Therefore, the shapes of the streamlines change from circular to elliptical forms. Hence, the alteration provokes the heat transfer process in the cavity to ameliorate or reduce.

IV. CONCLUSIONS

In this paper, natural convection of Al_2O_3 water nanofluids in a cubic cavity in the presence of magnetic field are numerically investigated. The effects of nanoparticles volume fraction Rayleigh number, Hartmann number, and inclination angle on the flow and heat transfer characteristics have been examined. The following conclusions can be drawn.

1. Adding nanoparticles of into pure water improves natural convection heat transfer in a cubic cavity.
2. In contrast to Rayleigh number, the increase of Hartmann number decreases the heat transfer rate. This effect is more pronounced at high Rayleigh numbers.
3. The influences of external magnetic field on heat transfer vary with different orientations.

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